

FEM勉強会(第4回)

その2、はり、平板、シェル解析について

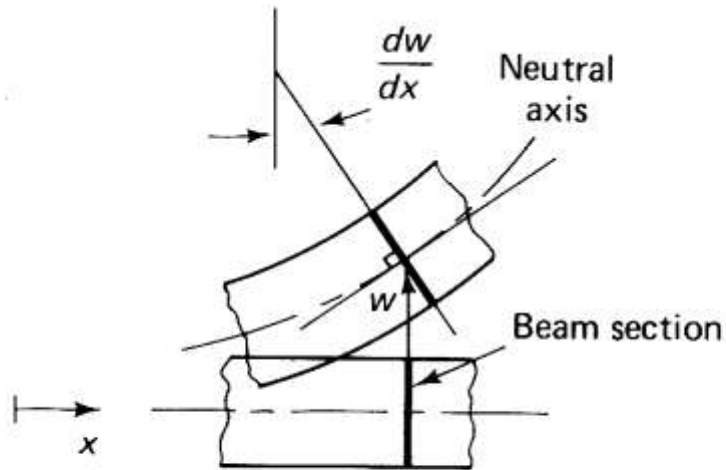
平成22年11月10日

園田 恵一郎

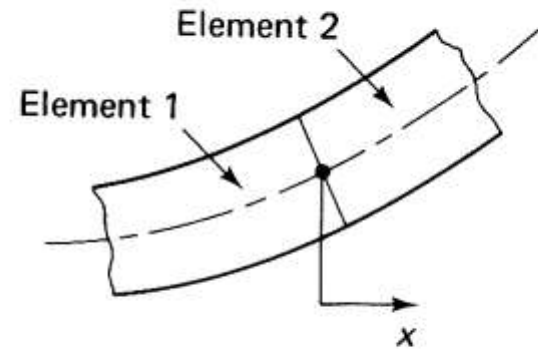


はりの理論について

Bernoulli-Eulerはり理論



Deformation of cross-section



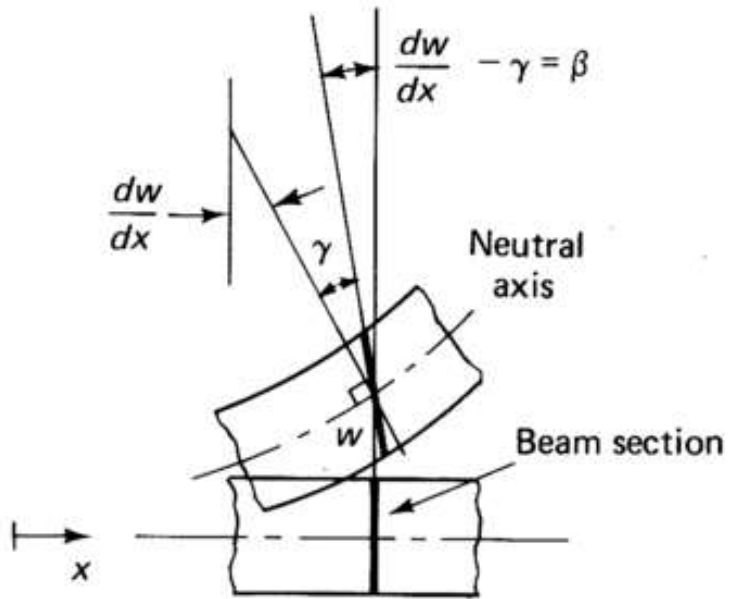
Boundary conditions between beam elements

$$w \Big|_{x^{-0}} = w \Big|_{x^{+0}} ; \frac{dw}{dx} \Big|_{x^{-0}} = \frac{dw}{dx} \Big|_{x^{+0}}$$

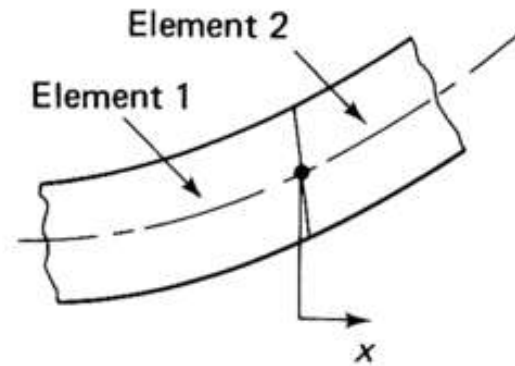
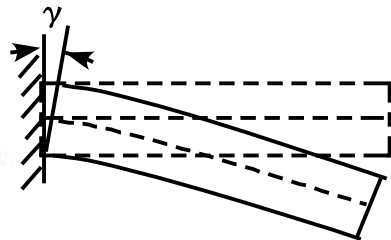
(a) Beam deformations excluding shear effect



Timoshenkoはり理論



Deformation of cross-section



Boundary conditions between beam elements

$$w \Big|_{x^{-0}} = w \Big|_{x^{+0}}$$

$$\beta \Big|_{x^{-0}} = \beta \Big|_{x^{+0}}$$



Bernoulli-Eulerはり理論

w: たわみ、 たわみ角: $\theta = \frac{dw}{dx}$ 曲率: $\phi = \frac{d\theta}{dx} = \frac{d^2w}{dx^2}$

弾性則: $M = EI\phi$ $Q = \frac{dM}{dx}$

Timoshenkoはり理論

せん断変形: γ 回転角: $\beta = \frac{dw}{dx} - \gamma$ 曲率: $\phi = \frac{d\beta}{dx}$,

弾性則: $\tau = \frac{Q}{\kappa A}$ $\kappa = 5/6$ $\gamma = \frac{\tau}{G}$

$$M = EI\phi = EI \frac{d\beta}{dx} = EI \left(\frac{d^2w}{dx^2} - \frac{d\gamma}{dx} \right)$$



ひずみエネルギー

$$W_i = \frac{EI}{2} \int_0^L \phi^2 dx + \frac{GA\kappa}{2} \int_0^L \gamma^2 dx$$

ポテンシャル
エネルギーと

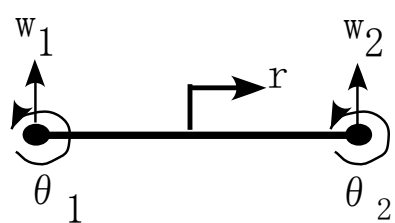
$$\pi = \frac{EI}{2} \int_0^L \left(\frac{d\beta}{dx} \right)^2 dx + \frac{GA\kappa}{2} \int_0^L \left(\frac{dw}{dx} - \beta \right)^2 dx - \int_0^L p w dx$$

極小の原理

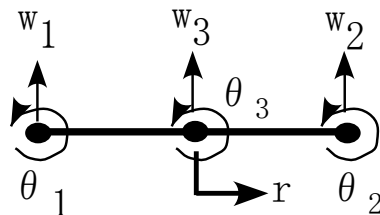
$$EI \int_0^L \left(\frac{d\beta}{dx} \right) \cdot \delta \left(\frac{d\beta}{dx} \right) dx + GA\kappa \int_0^L \left(\frac{dw}{dx} - \beta \right) \cdot \delta \left(\frac{dw}{dx} - \beta \right) dx - \int_0^L p \cdot \delta w dx = 0$$

補間関数

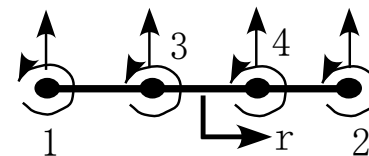
$$w = \sum_{i=1}^q h_i w_i \quad \beta = \sum_{i=1}^q h_i \theta_i$$



(a) q=2



(b) q=3



(c) q=4



FEMの定式化

$$w = \mathbf{H}_w \hat{\mathbf{u}} \quad \beta = \mathbf{H}_\beta \hat{\mathbf{u}}$$

$$\hat{\mathbf{u}} = \begin{bmatrix} w_1 & \dots & w_q & \theta_1 & \dots & \theta_q \end{bmatrix}$$

$$\mathbf{H}_w = \begin{bmatrix} h_1 & \dots & h_q & 0 & \dots & 0 \end{bmatrix}^T \quad \mathbf{H}_\beta = \begin{bmatrix} 0 & \dots & 0 & h_1 & \dots & h_q \end{bmatrix}^T$$

$$\frac{\partial w}{\partial x} = \mathbf{B}_w \hat{\mathbf{u}} \quad \frac{\partial \beta}{\partial x} = \mathbf{B}_\beta \hat{\mathbf{u}}$$

$$\mathbf{B}_w = \mathbf{J}^{-1} \begin{bmatrix} \frac{\partial h_1}{\partial r} & \dots & \frac{\partial h_q}{\partial r} & 0 & \dots & 0 \end{bmatrix} \quad \mathbf{B}_\beta = \mathbf{J}^{-1} \begin{bmatrix} 0 & \dots & 0 & \frac{\partial h_1}{\partial r} & \dots & \frac{\partial h_q}{\partial r} \end{bmatrix}$$

$$\mathbf{K} = EI \int_{-1}^1 \mathbf{B}_\beta^T \mathbf{B}_\beta \det \mathbf{J} + GA\kappa \int_{-1}^1 (\mathbf{B}_w - \mathbf{H}_\beta)^T (\mathbf{B}_w - \mathbf{H}_\beta) \det \mathbf{J} dr$$

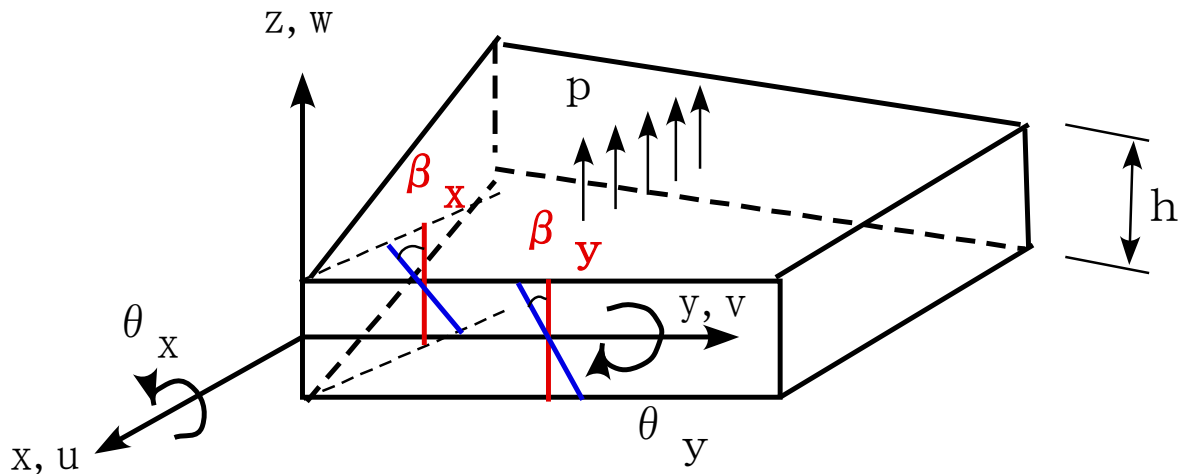
$$\mathbf{R} = \int_{-1}^1 \mathbf{H}_w^T p \det \mathbf{J} dr \quad \text{剛性方程式:} \quad \mathbf{K} \hat{\mathbf{u}} = \mathbf{R}$$



平板のFEM解析

$$u = z\beta_x(x, y) \quad v = -z\beta_y(x, y) \quad w = w(x, y)$$

$$\begin{bmatrix} \varepsilon_x \\ \varepsilon_y \\ \gamma_{xy} \end{bmatrix} = z \begin{bmatrix} \partial\beta_x / \partial x \\ -\partial\beta_y / \partial y \\ \partial\beta_x / \partial y - \partial\beta_y / \partial x \end{bmatrix} \quad \begin{bmatrix} \gamma_{yz} \\ \gamma_{xz} \end{bmatrix} = \begin{bmatrix} \partial w / \partial y - \beta_y \\ \partial w / \partial x + \beta_x \end{bmatrix}$$



β_x β_y 回転角



FEMの定式化

$$\begin{bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{bmatrix} = z \cdot \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & (1-\nu)/2 \end{bmatrix} \cdot \begin{bmatrix} \partial\beta_x / \partial x \\ -\partial\beta_y / \partial y \\ \partial\beta_x / \partial y - \partial\beta_y / \partial x \end{bmatrix}$$

$$\begin{bmatrix} \tau_{yz} \\ \tau_{xz} \end{bmatrix} = \frac{E}{2(1+\nu)} \begin{bmatrix} \partial w / \partial y - \beta_y \\ \partial w / \partial x + \beta_x \end{bmatrix}$$

$$\boldsymbol{\varphi} = \begin{bmatrix} \partial\beta_x / \partial x \\ -\partial\beta_y / \partial y \\ \partial\beta_x / \partial y - \partial\beta_y / \partial x \end{bmatrix}$$

$$\pi = \frac{1}{2} \int_{A_0-h/2}^{h/2} \int \begin{bmatrix} \varepsilon_x & \varepsilon_y & \gamma_{xy} \end{bmatrix} \cdot \begin{bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{bmatrix} \cdot dz dA_0 + \frac{\kappa}{2} \int_{A_0-h/2}^{h/2} \int \begin{bmatrix} \gamma_{yz} & \gamma_{xz} \end{bmatrix} \cdot \begin{bmatrix} \tau_{yz} \\ \tau_{xz} \end{bmatrix} \cdot dz dA_0 - \int_{A_0} w p dA_0$$

$$\pi = \frac{1}{2} \int_{A_0} \boldsymbol{\varphi}^T \mathbf{C}_b \boldsymbol{\varphi} dA_0 + \frac{1}{2} \int_{A_0} \boldsymbol{\gamma}^T \mathbf{C}_s \boldsymbol{\gamma} dA_0 - \int_{A_0} w p dA_0$$

$$\boldsymbol{\gamma} = \begin{bmatrix} \partial w / \partial y - \beta_y \\ \partial w / \partial x + \beta_x \end{bmatrix}$$



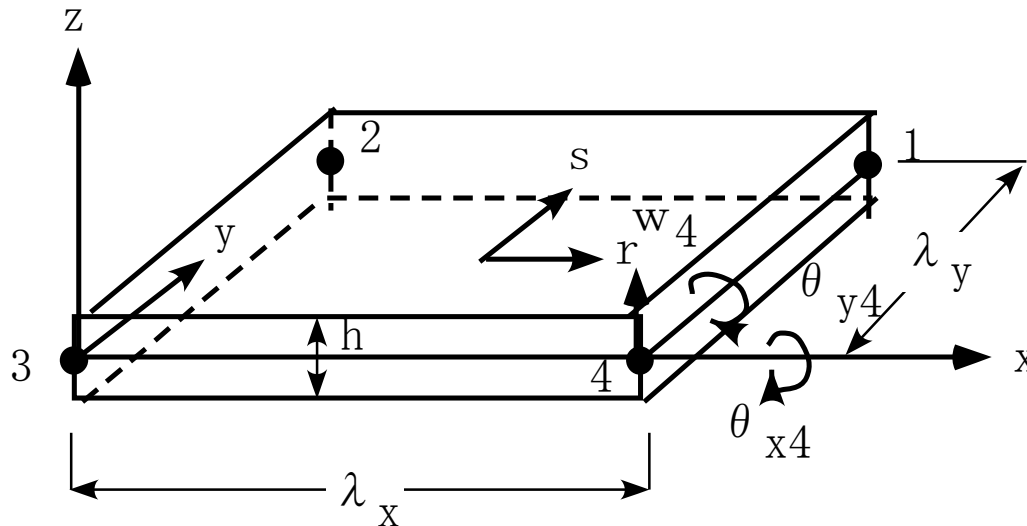
$$\hat{\mathbf{u}} = [w_1 \quad \theta_{x1} \quad \theta_{y1} \quad w_2 \dots \dots \theta_{y4}]^T$$

ポテンシャルエネルギー極小の条件

$$\int_{A_0} \delta \boldsymbol{\varphi}^T \mathbf{C}_b \boldsymbol{\varphi} dA_0 + \int_{A_0} \delta \boldsymbol{\gamma}^T \mathbf{C}_s \boldsymbol{\gamma} dA_0 - \int_{A_0} \delta w p dA_0 = 0$$

$$\boldsymbol{\varphi}(r, s) = \mathbf{B}_\phi \hat{\mathbf{u}} \quad \boldsymbol{\gamma}(r, s) = \mathbf{B}_s \hat{\mathbf{u}} \quad w(r, s) = \mathbf{H}_w \hat{\mathbf{u}}$$

$$\hat{\mathbf{u}} = [w_1 \quad \theta_{x1} \quad \theta_{y1} \quad w_2 \dots \dots \theta_{y4}]^T$$



$$h_1 = \frac{1}{4}(1+r)(1+s) \quad h_2 = \frac{1}{4}(1-r)(1+s) \quad h_3 = \frac{1}{4}(1-r)(1-s) \quad h_4 = \frac{1}{4}(1+r)(1-s)$$



剛性方程式 $\mathbf{K}\hat{\mathbf{u}} = \mathbf{R}_s$

$$\mathbf{K} = \int_{-1}^1 \int_{-1}^1 (\mathbf{B}_\phi^T \mathbf{C}_b \mathbf{B}_\phi + \mathbf{B}_s^T \mathbf{C}_s \mathbf{B}_s) \det \mathbf{J} \cdot dr ds$$

$$\mathbf{R}_s = \int_{-1}^1 \int_{-1}^1 \mathbf{H}_w^T p \det \mathbf{J} \cdot dr ds$$

境界条件:

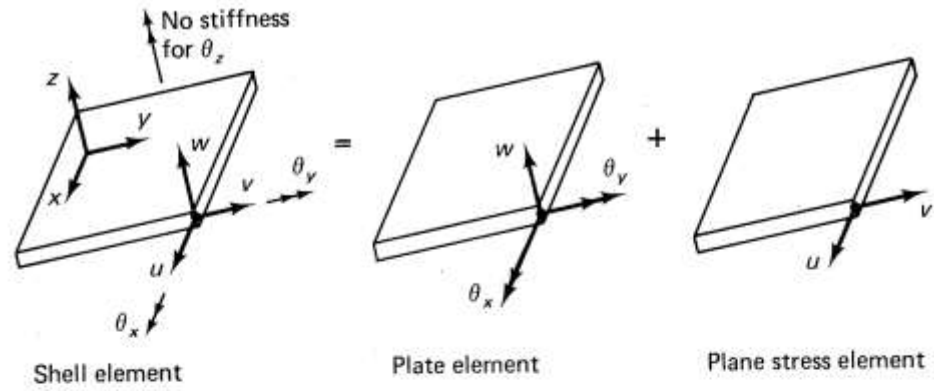
単純支持辺: $w = 0$

固定辺: $w = 0 \quad \theta_x = \beta_x = 0, \theta_y = \beta_y = 0$

自由辺: なし

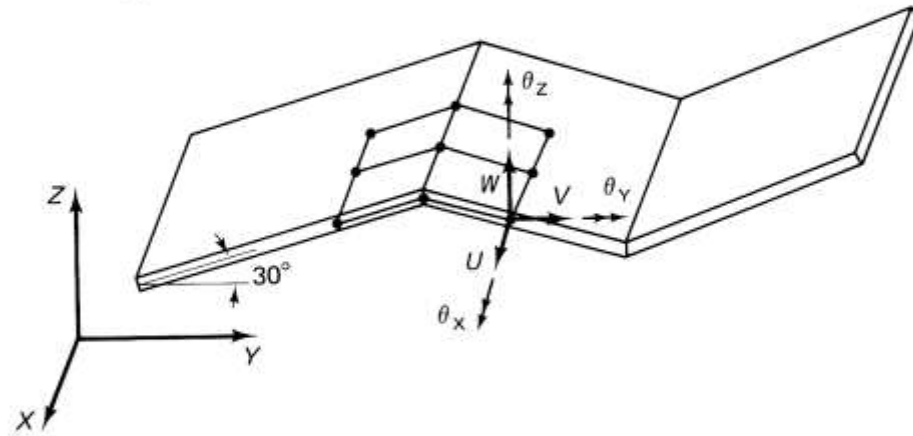


シェルの FEM解析

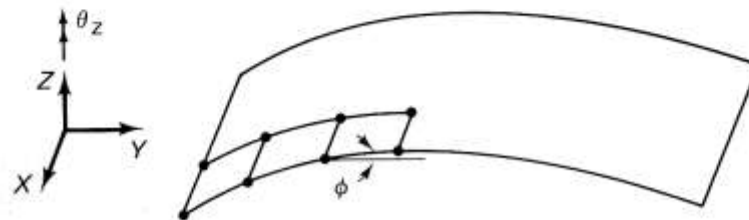


(a) Basic shell element with local 5 degrees of freedom at a node

要素: 板曲げ + シャイベ



(b) Analysis of folded plate structure



(c) Analysis of slightly curved shell

FIGURE 4.19 Use of a flat shell element.



四辺形要素の場合

$$\hat{\mathbf{K}}_{shell} = \begin{bmatrix} \mathbf{K}_B & \mathbf{0} \\ \mathbf{0} & \mathbf{K}_M \end{bmatrix}$$

20x20

$$\mathbf{K}_{shell} = \mathbf{T}^T \hat{\mathbf{K}}_{shell} \mathbf{T}$$

24x24

$$\mathbf{K}_{shell} = \begin{bmatrix} \hat{\mathbf{K}}_{shell} & \mathbf{0} \\ \mathbf{0} & k\mathbf{I} \end{bmatrix}$$

24x24

